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CALIFORNIA STATE UNIVERSITY, LOS ANGELES

Jet Propulsion Laboratory
California Institute of Technology

Project 6:
Venus Wind Harvester

Group Members:

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Kevin Pan
Oscar Lopez
Zacharias Garza
Jonathan Serrano
Anthony Izaguirre

Faculty Adviser:

Dr. Jim Kuo

Client

Dr. Jonathan Sauder

Agenda

1. Introduction	Saul Loza
2. Methodology	Kevin Pan
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4. Results & Validation	Zacharias, Anthony & Kevin
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6. Conclusion	Anthony Izaguirre
7. Future Plans	Saul Loza

Introduction

- Why Venus?
- Earth’s sister-planet
- Relatively unknown
- Possible microbial life

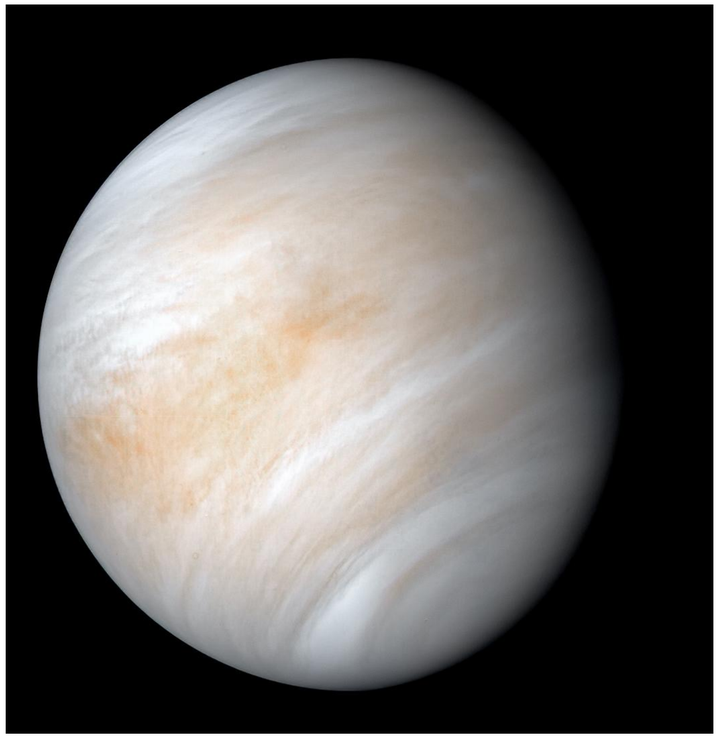


Figure 2: Image of Venus from outer space.

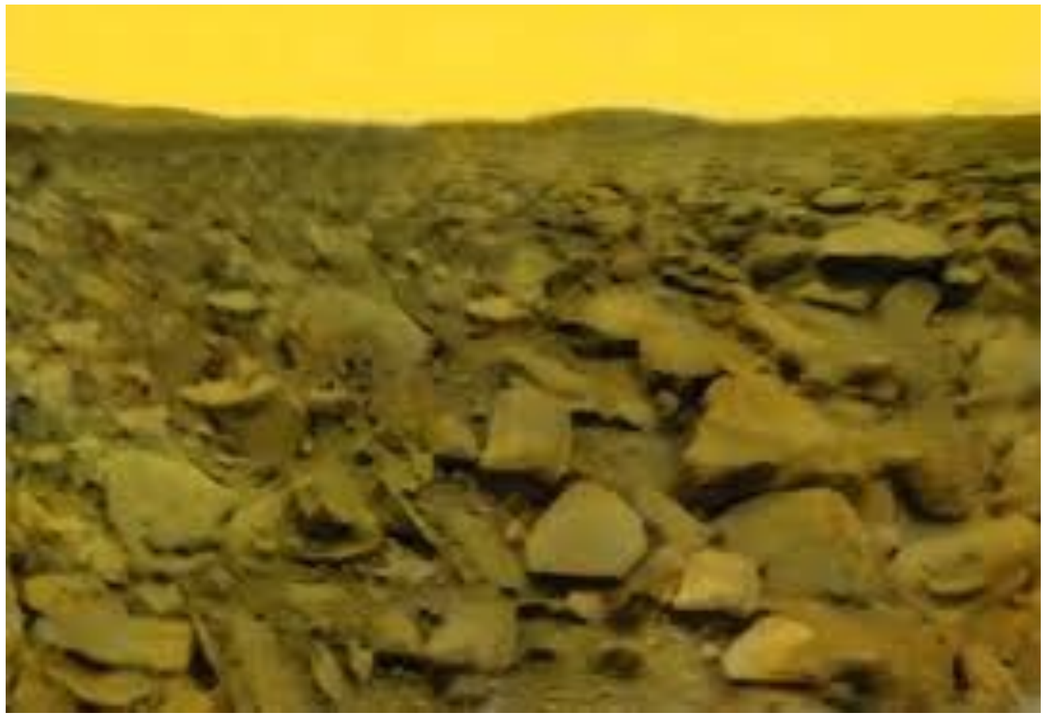


Figure 1: Image of Venus from the surface

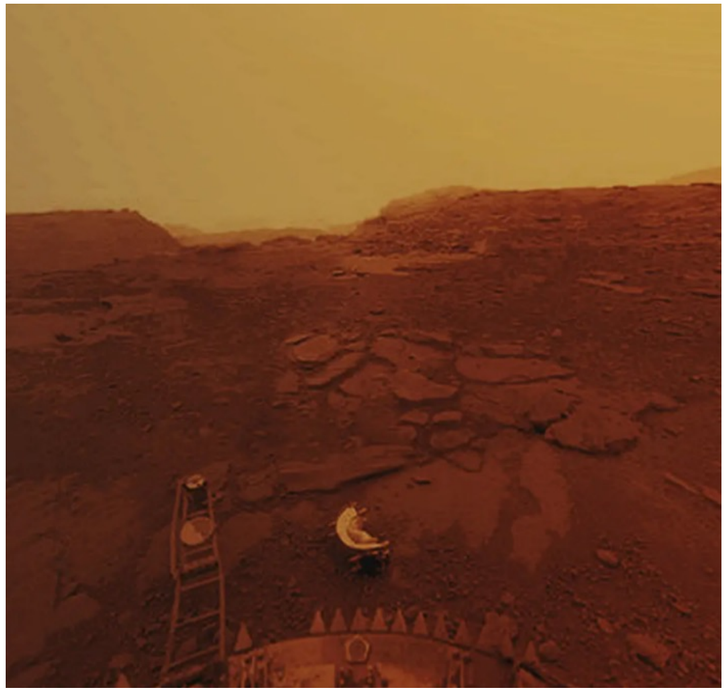


Figure 3: Image of Venus from the surface

NASA/JPL Venus Mission: AREE



Figure 4: Rendering of possible Venus Rover Design [1]

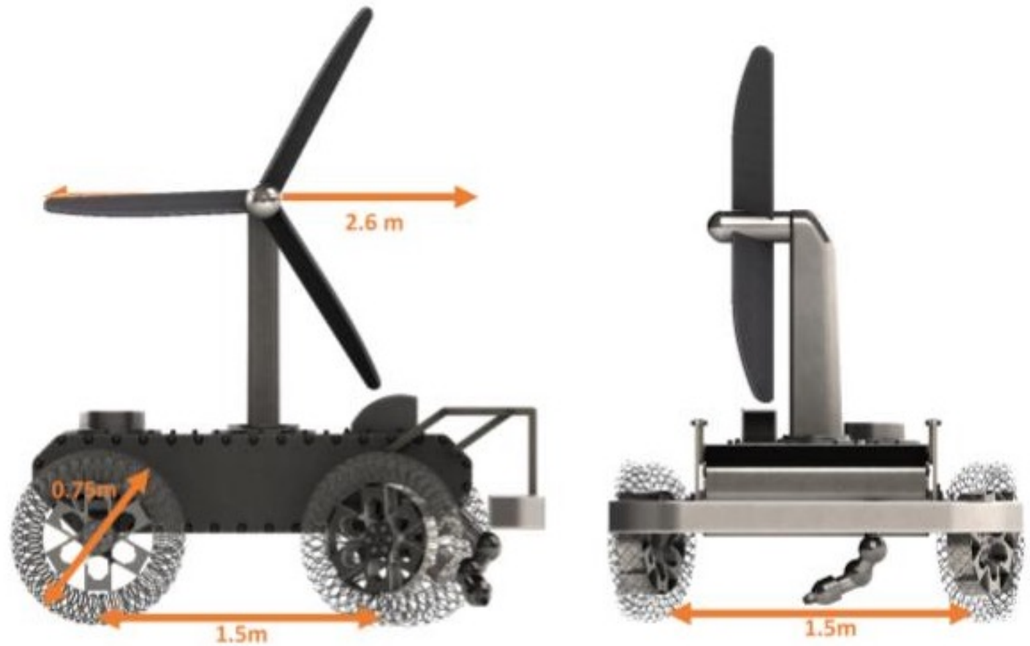


Figure 5: Front and side view of possible Venus Rover Design. [2]

- Completely mechanical system
- Venus average surface temperature$\approx 425℃$
- Alternate energy source: Wind Turbine

Objective/Project Requirements

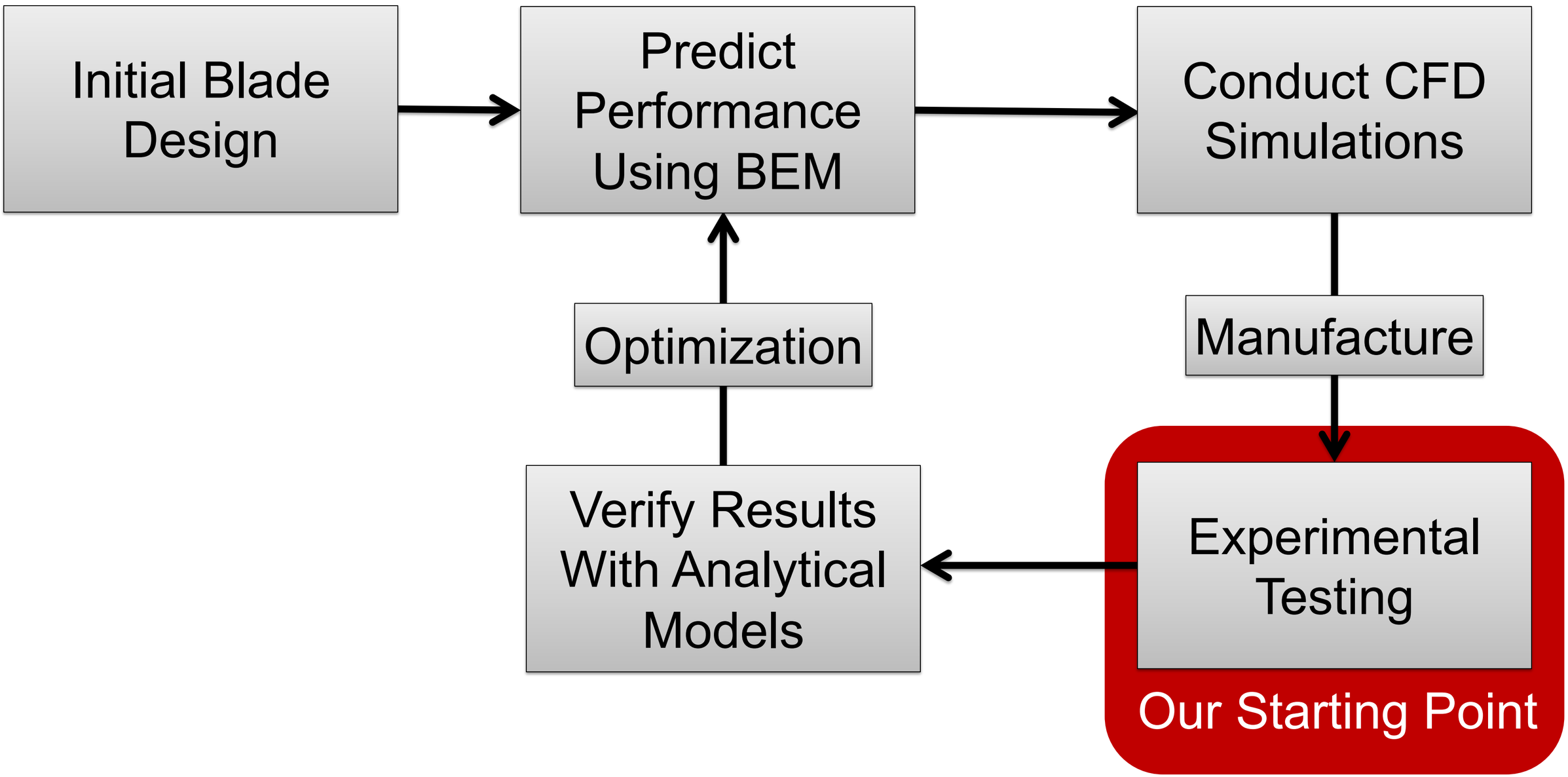
Table 1: Project Requirements and Parameters

Requirements	Requirement Values
Storing Cylinder Constraints	2.7 m (diameter) x 0.5 m (thickness)
Mass	< 45 kg
Power Generated at 0.6 $\frac{m}{s}$ (Venus)	9 W
Efficiency	≥ 40 %
Wind Speed Range (Venus)	(0.3 $\frac{m}{s}$ – 1.3 $\frac{m}{s}$)
Budget	< $1,000

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Methodology



Methodology

- Fourth Year of Project
- Past Team's Work
  - Turbine Design Methodology
  - Turbine Design Optimization
  - Wind Tunnel Testing
  - Preliminary Water Testing

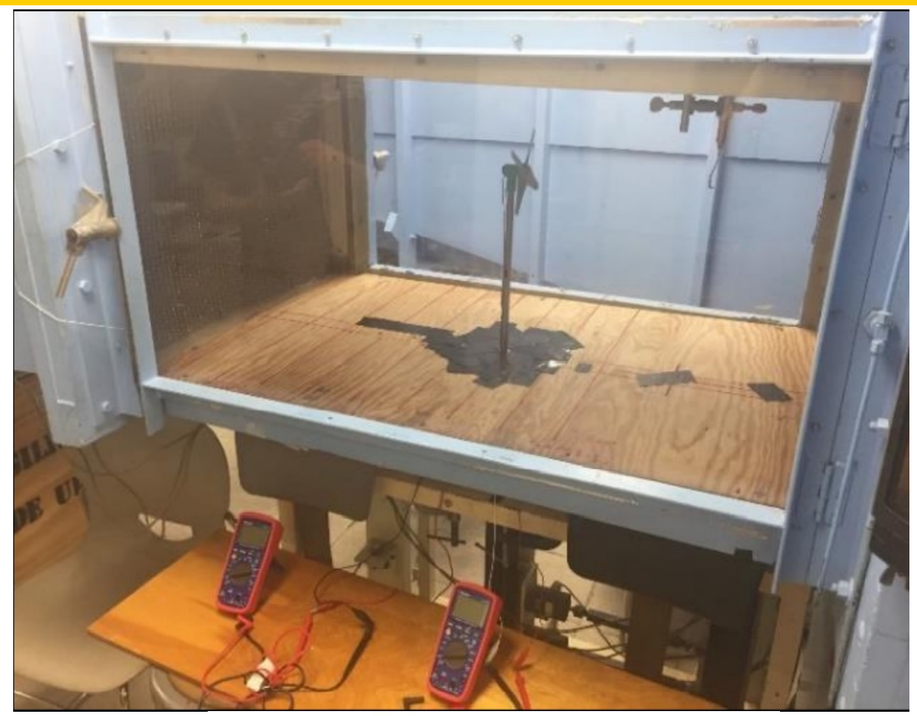


Figure 6: (Top) wind tunnel testing,(Bottom) 19-20 CAD design

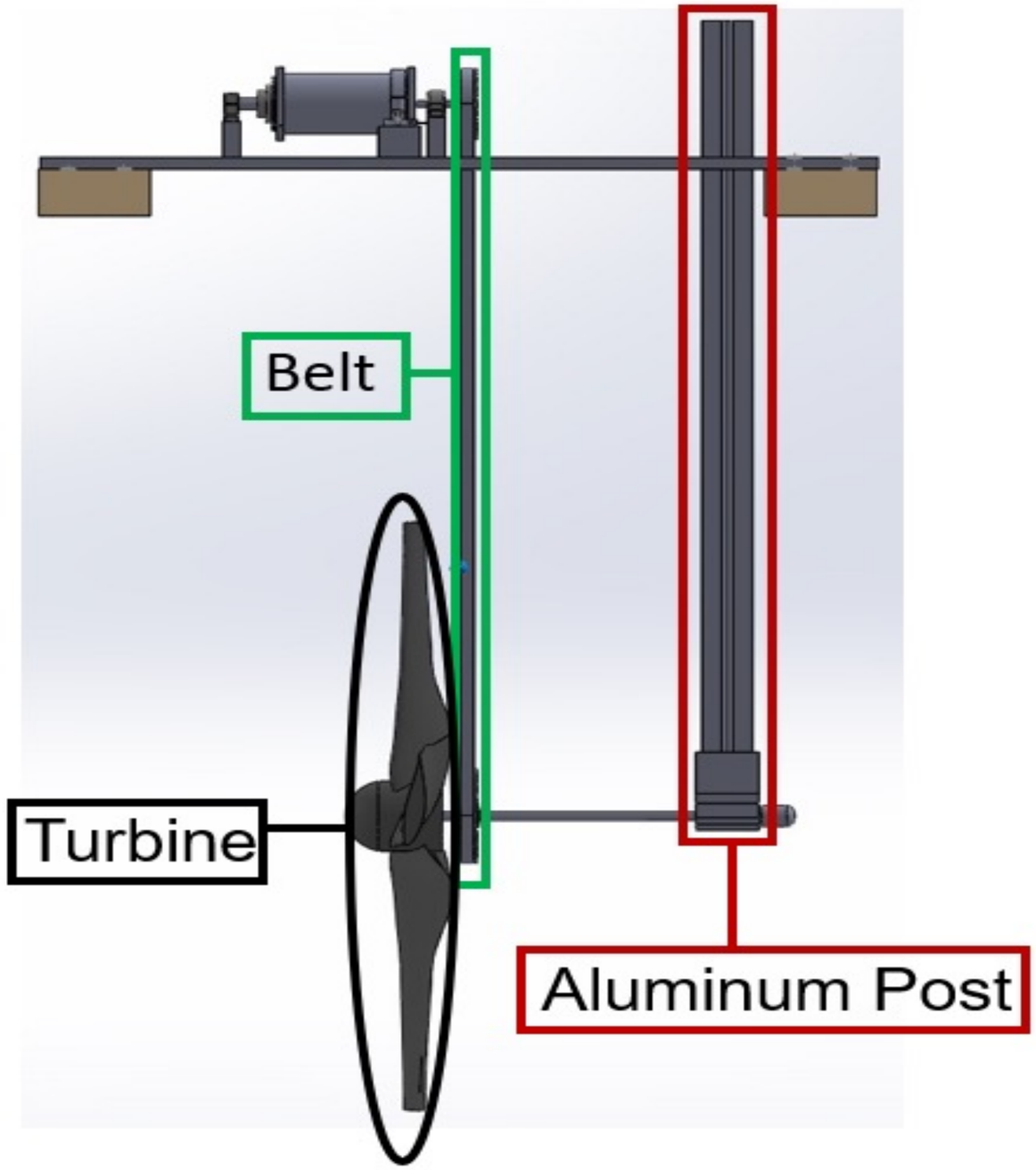
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2020-2021 Test Requirements

- Previous Design Problems
  - Fluid velocity
  - Stationary
- Method to measure fluid velocity
- Method to measure torque
- Method to pulley rpm

CAD Design



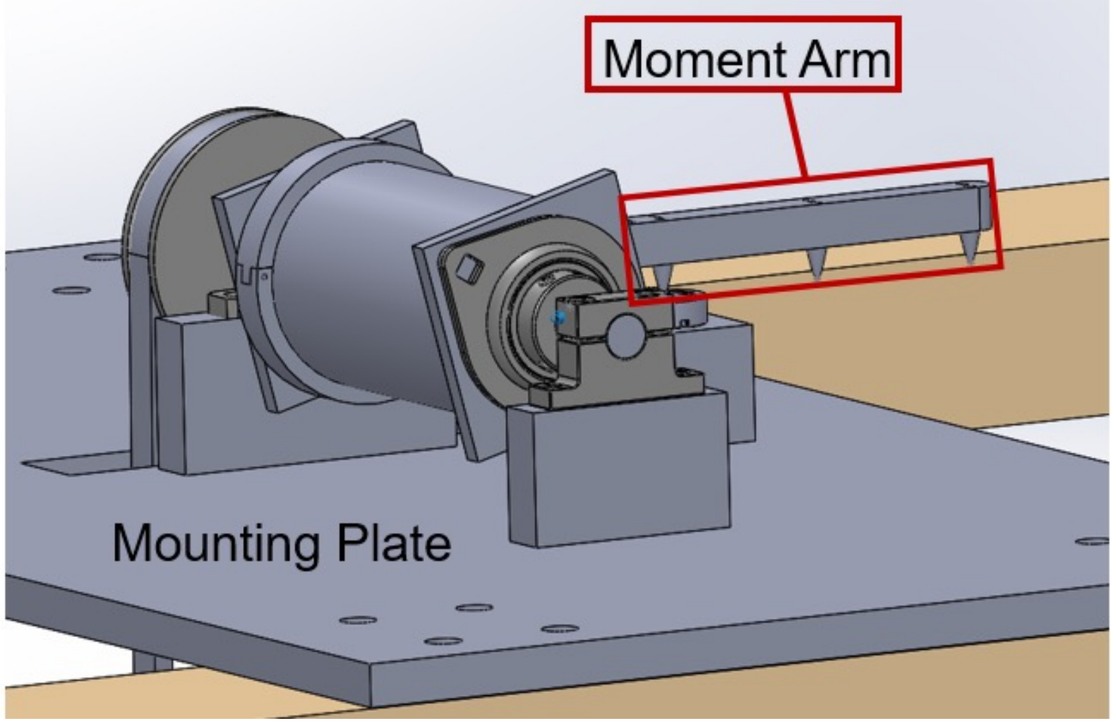
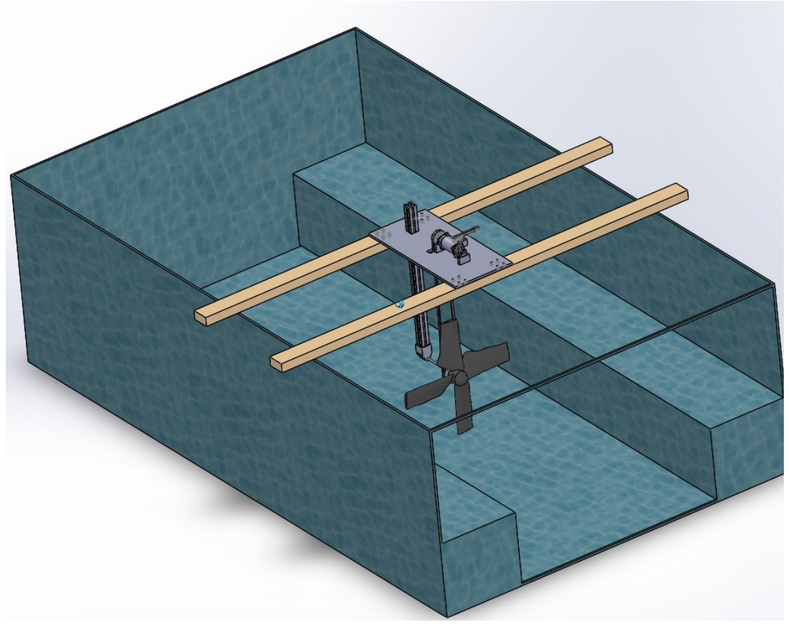


Figure 7: (Left) Side view of winter testing CAD design, (Top right) Side view of mounting plate CAD design, (Bottom right) Rear view of mounting plate CAD design.

CAD vs Testing Design

- This year's test set up CAD



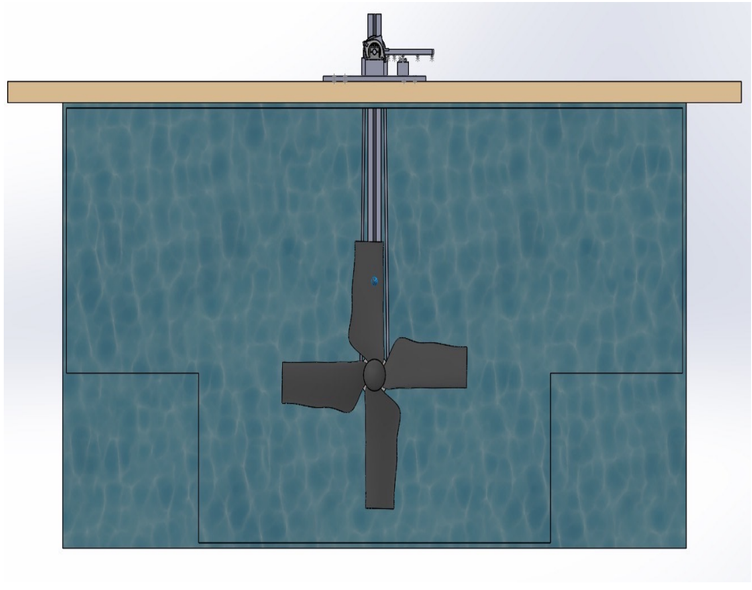
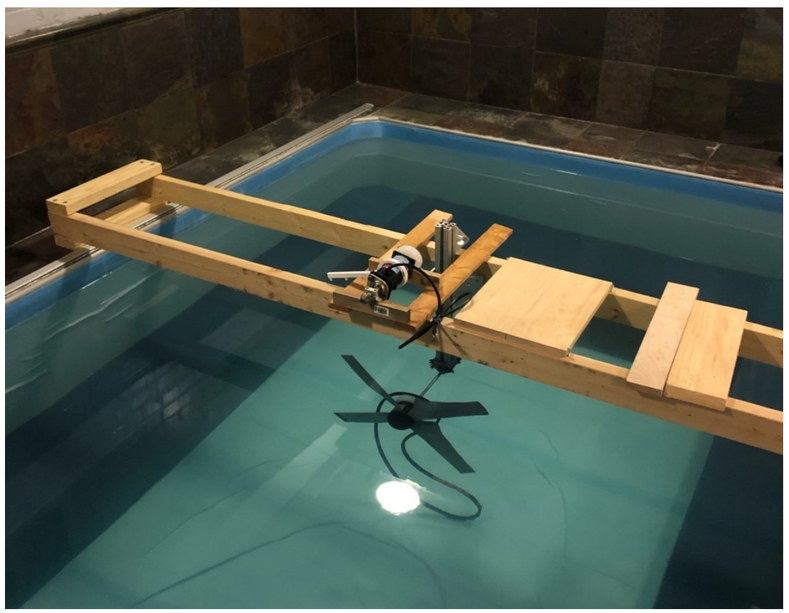


Figure 8: CAD design of turbine placement

- Stationary design during testing



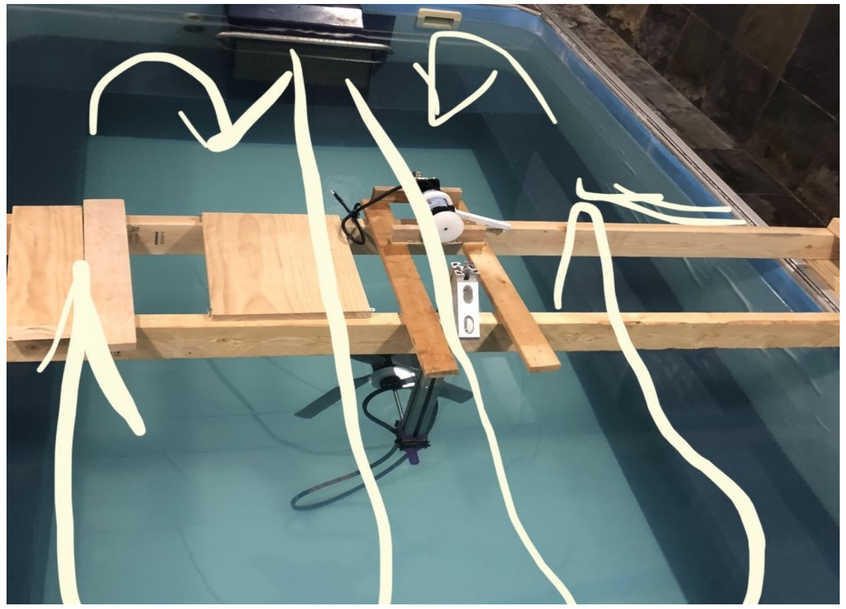
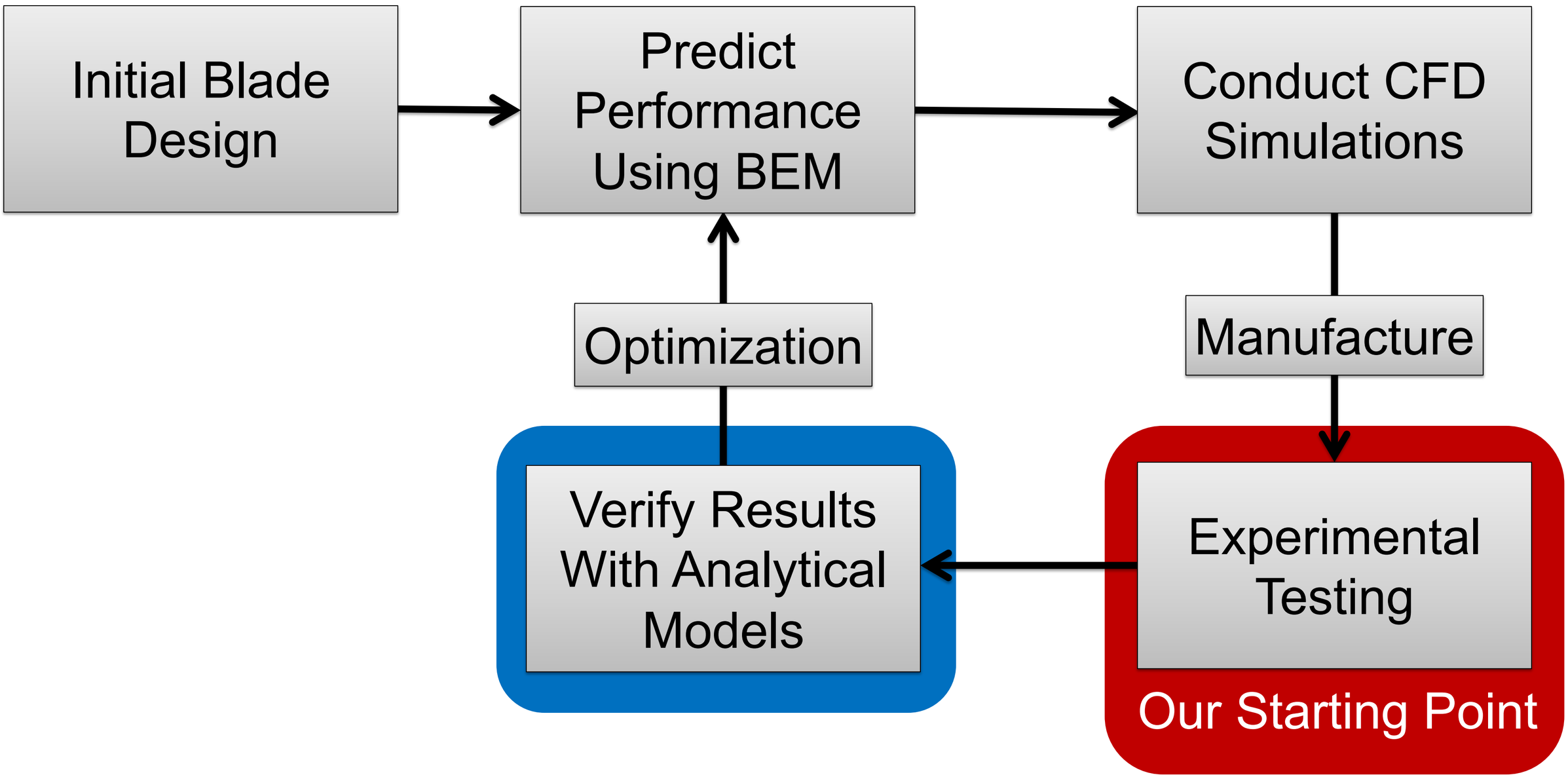


Figure 9: Winter testing setup and placement of turbine

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Methodology



Tip Speed Ratio

$$\text{TSR} = \frac{\text{Tip Rotational Speed}}{\text{Velocity of Wind}}$$

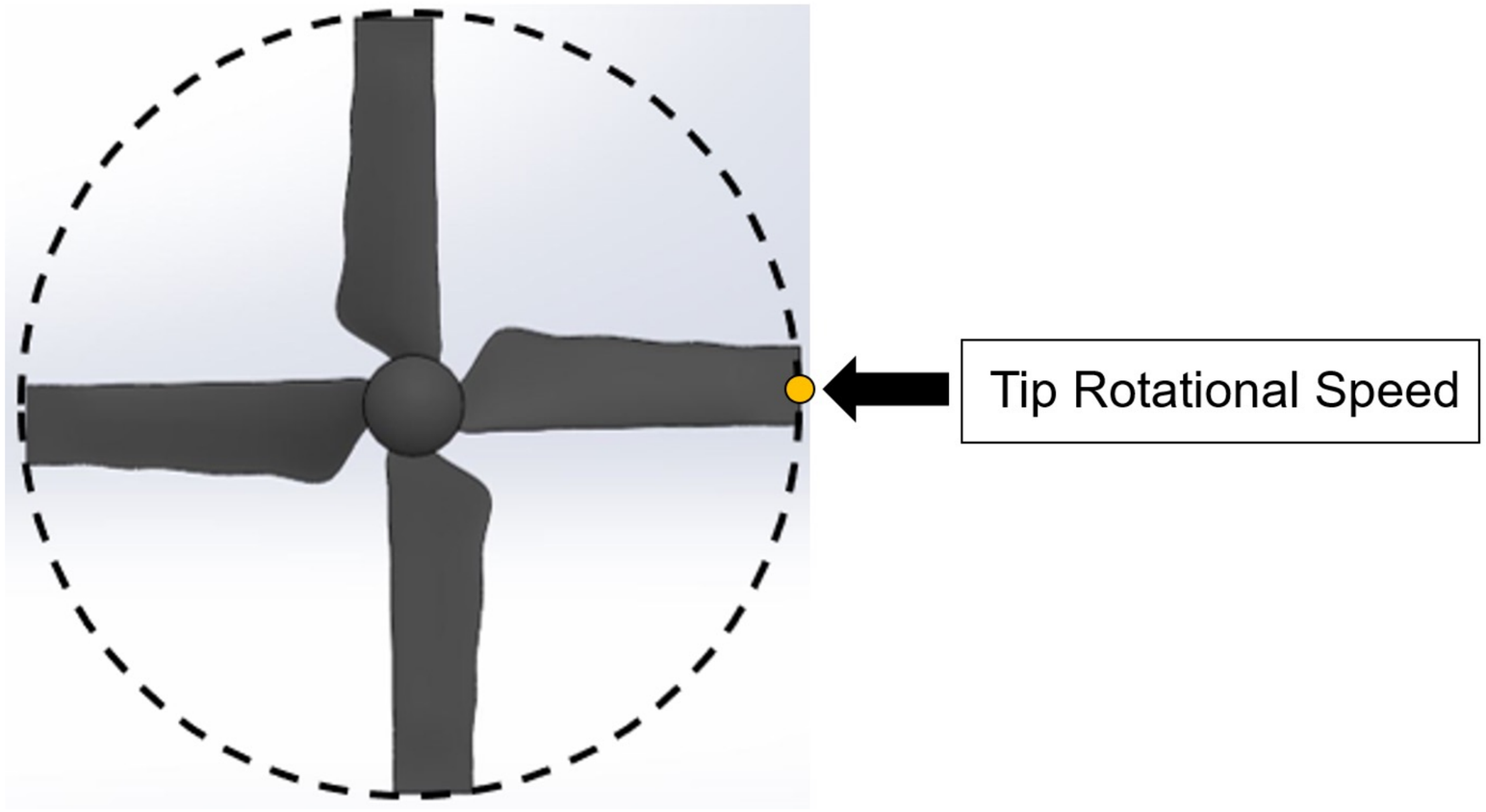


Figure 10: Definition of tip speed ratio

Results

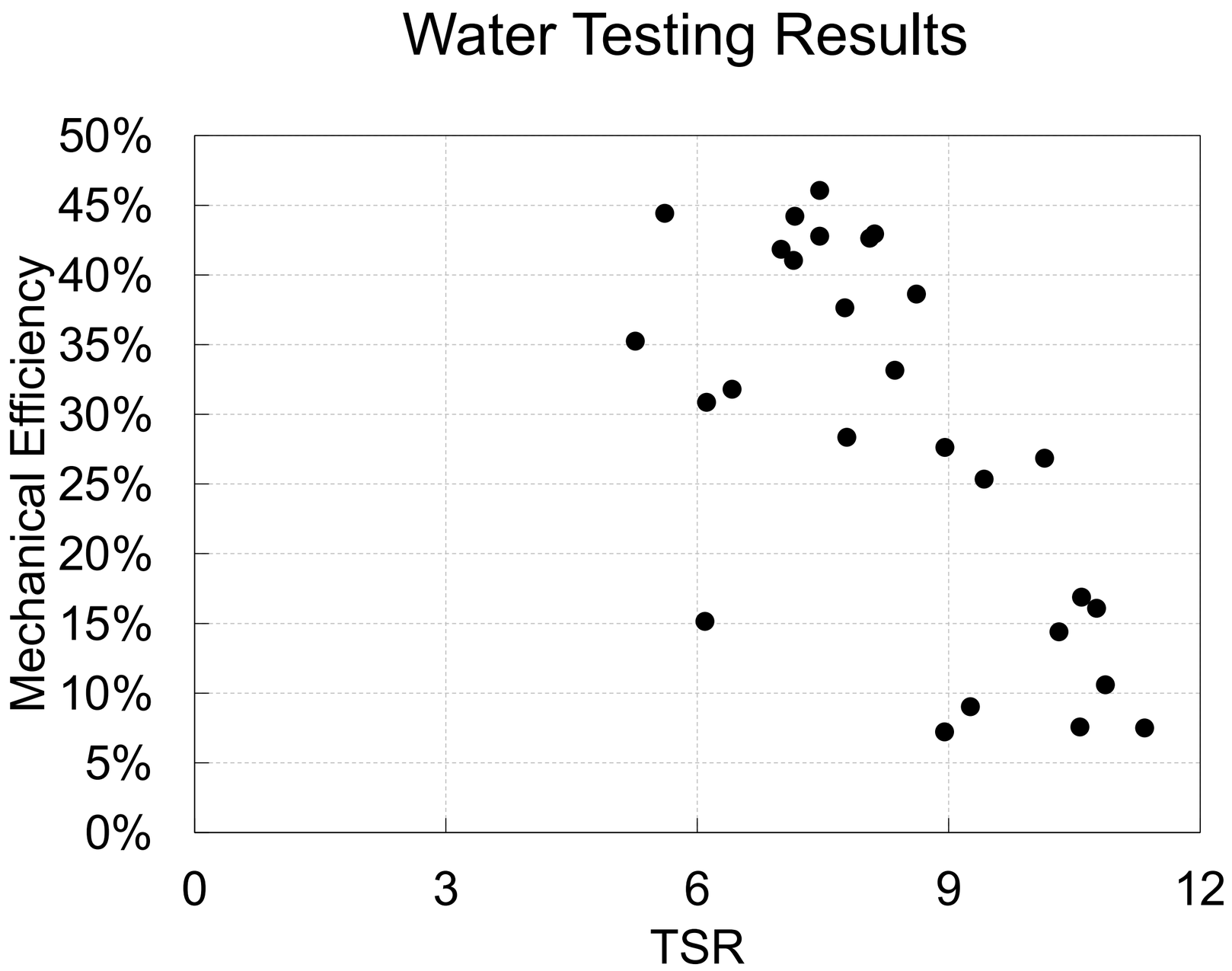


Figure 10: Mechanical efficiency based on TSR values obtained during winter testing

Blade Element Momentum Theory (BEM)

Relates blade shape to the rotor's ability to extract power from the wind

- Analysis of forces at section of blade as a function of blade geometry
  - Expressed as a function of lift and drag coefficients and angle of attack (angle between chord and relative wind)
- Defines the normal (thrust) and tangential (torque) forces on the annular rotor section as a function of the flow angles

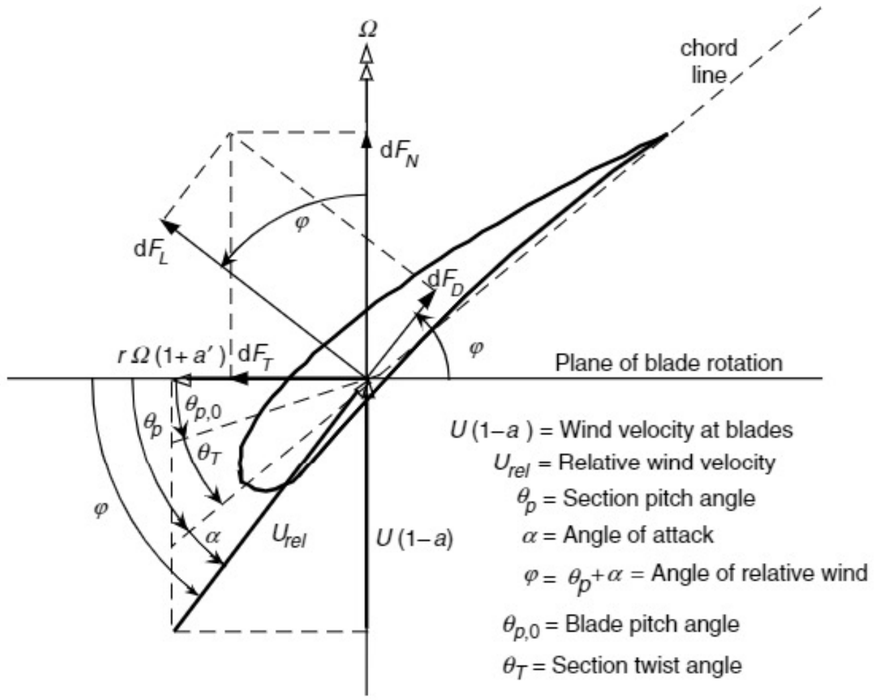


Figure 11: Blade geometry for analysis of a horizontal axis wind turbine.

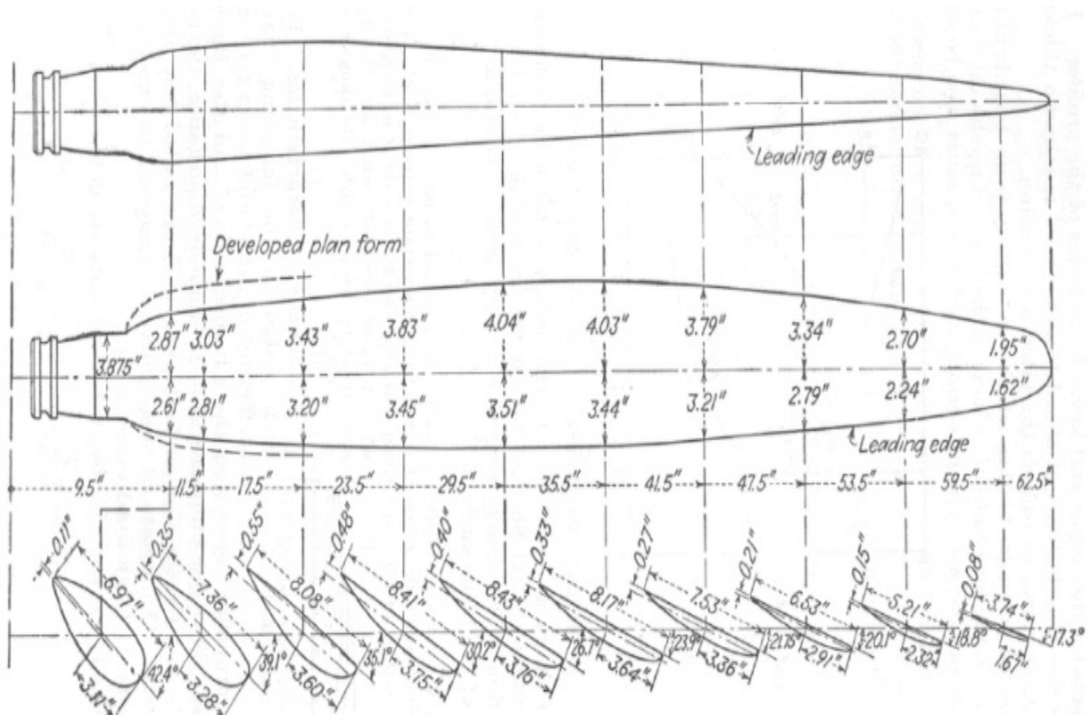


Figure 12: Sectioned blade of a horizontal axis wind turbine.

BEM Analysis

- Matlab code to perform calculations based on BEM theory to understand expected results from experimental testing
- Used to validate experimental data

Table 2: Theoretical turbine output from BEM theory compared to experimentally observed turbine output during water testing

Experimental				Theoretical			
Average Fluid Velocity (m/s)	RPM	Mechanical Efficiency	Power (W)	Mechanical Efficiency	Percent Difference	Power (W)	Percent Difference
0.53	120	0.442	9.852	0.319	38.51%	7.907	24.59%
	125	0.461	10.263	0.313	47.18%	7.751	32.41%
	125	0.428	9.529	0.313	36.67%	7.751	22.94%
	130	0.376	8.386	0.304	23.91%	7.523	11.48%
	135	0.426	9.500	0.292	45.90%	7.239	31.24%
	140	0.332	7.389	0.275	20.44%	6.820	8.34%
	150	0.276	6.158	0.207	33.85%	5.110	20.51%
	170	0.269	5.982	-0.819	132.78%	-20.285	128.24%

Computational Fluid Dynamics (CFD) Analysis

- CFD – Ansys Fluent Software
  - Viscosity Model: SST k-ω
  - Control Volume setup
  - Mesh Generation

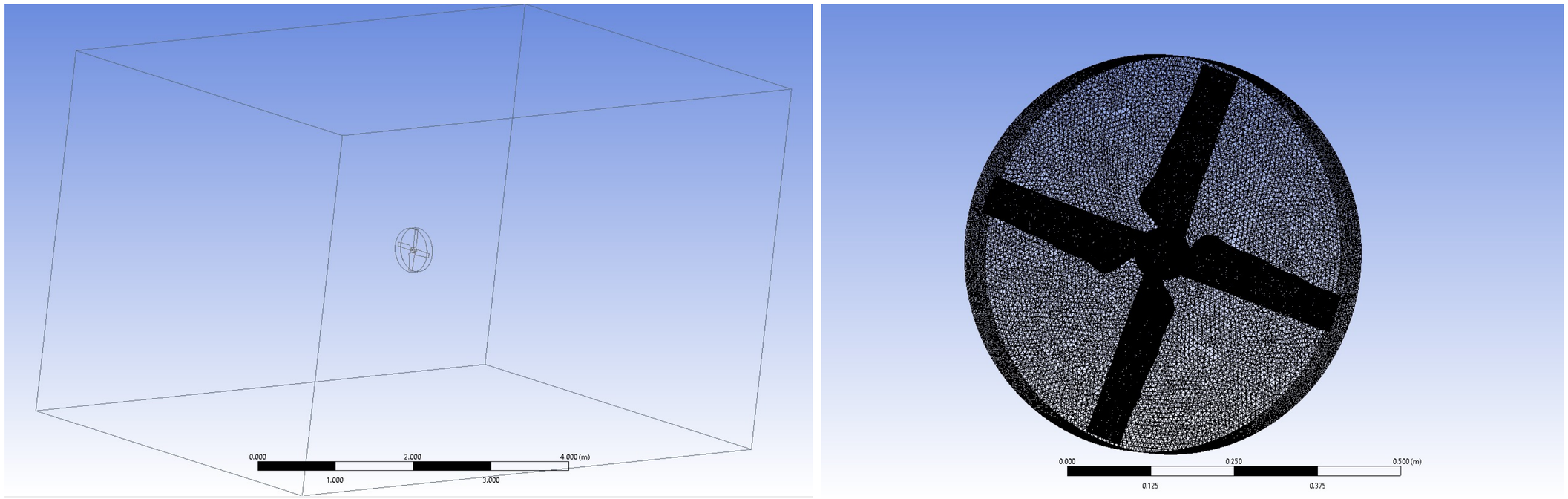


Figure 13: Current model Setup and Tetrahedra Mesh on Turbine

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Improvements on Test Setup

- Winter Testing Had Issues
  - Uncharacterized motor
  - Pulley slipping
  - Wrong size shaft bearing
- Refinement Necessary
  - Update data acquisition
  - Slipping issue corrected
  - New characterized motor
  - New motor housing
  - Updated moment arm assembly







Figure 14: (Top) Data acquisition, (Center) , (Bottom) New motor.

Improvements To The Test Setup

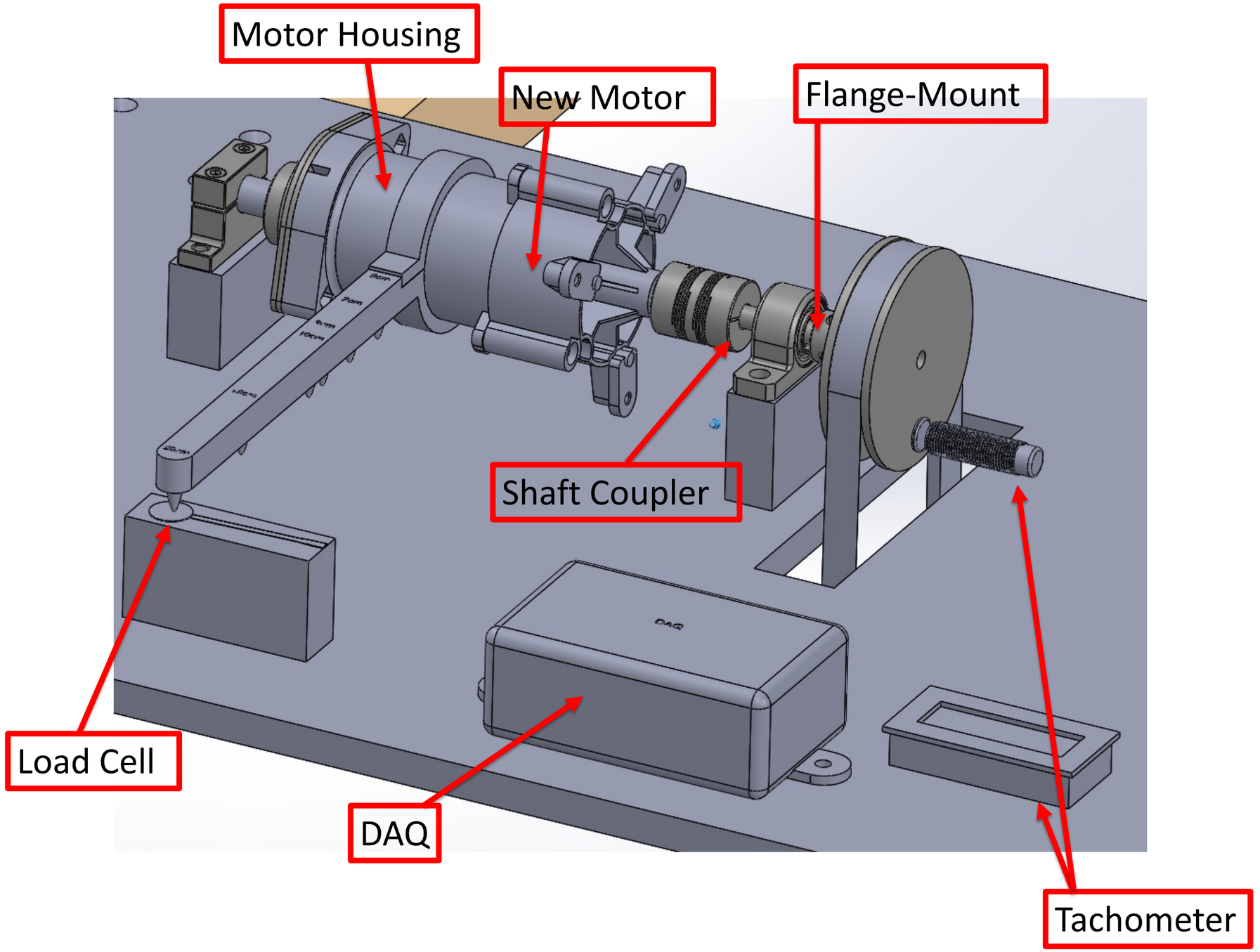


Figure 15: CAD design highlighting the improvements made based on the issues experienced during winter testing and the data obtained.

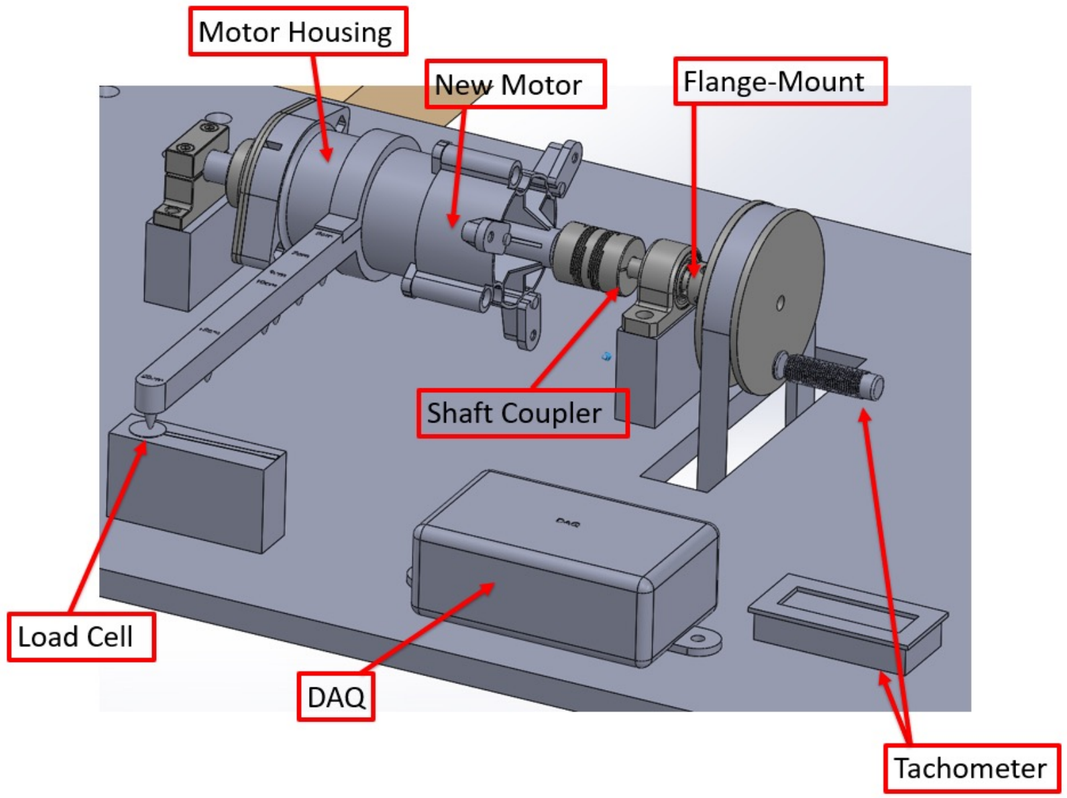
Final Bill Of Materials

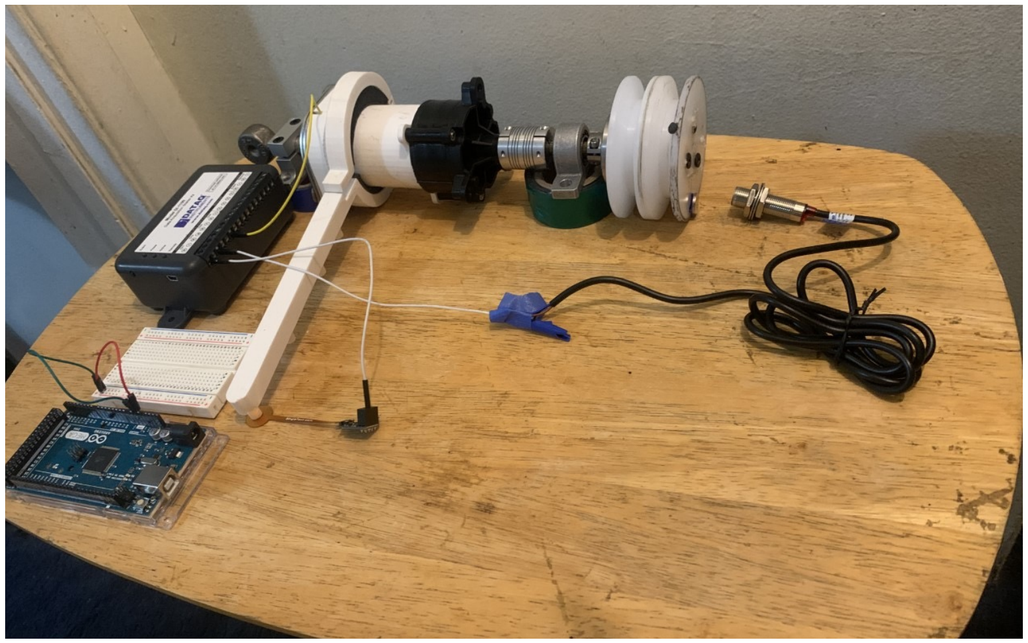
Table 3: Bill of materials for the senior design project of fall 20-21.

ITEM NO.	Materials	DESCRIPTION	QTY.	Price
1	(6383K215) Ball Bearing	Bearing Between Pulley and Hub	1	Inherited
2	Aluminum Post	Turbine Support	1	Inherited
3	DC Generator		1	Inherited
4	Generator Shaft		1	Inherited
5	MoldAdTSh	Connects turbine shafts to aluminum post	1	Inherited
6	(6274K21) Generator Pulley		1	Inherited
7	(6274K26) Turbine Pulley		1	Inherited
8	Turbine Blade and Hub		1	Inherited
9	Belt		1	Inherited
10	Single Tact Load Cell		1	Inherited
11	Breadboard		1	Inherited
12	Mounting Plate		1	Donated
13	Turbine Shaft		1	Donated
14	TwobyFour		2	Donated
15	(1865K3) Base Mounted Shaft Support	Supports generator rear shaft	1	Donated
16	Pool	Test Endless-pool	1	Donated
17	Wood Screws	Connects mounting plate to two by fours	16	Donated
18	Flow Meter Rental		1	Donated
19	Bearing Support	Roller Bearing Support	1	Fabricated
20	Moment Arm Bottom		1	Fabricated
21	Moment Arm Top		1	Fabricated
22	Shaft Bearing		1	27.94
23	Open Extended Gusset	5537T661	4	52.4
24	Single Nut with Button Head	5537T163	1	5.86
25	Rear motor mount	5913K71	1	11.33
26	Easy-Access Base	1865K3	1	18.45
27	Load Cell	FX1901-0001-0200	1	32.82
28	Data Acquisition	DI -1100	1	59
29	Rod End Bolt Blank	6065K321	1	12.37
30	Electrical Load	100W 1K ohm Potentiometer	1	27.36
31	Amplifier		1	21.99
32	Motor	Brushless DC Motor	1	23.99
33	Rotary Shaft Extender	1265K64	1	20.55
34	Flange mount	9624T16	1	75.42
35	Coupler	2463k28	1	51.13
36	Tacometer	Megnetic Tachometer	1	24.99
Total Cost				**293.51**

Updated Setup

- Updated Motor Assembly





- Updated CAD design made testbed build straight forward

Figure 16: Comparison between the improved CAD design and actual improved design.

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Conclusion

Table 4: Water Testing Result comparison to Requirement Values

Requirements	Requirement Values (Venus)	Performance Achieved In Water Testing
Storing Cylinder Constraints	2.7 m (diameter) x 0.5 m (thickness)	1:4 scale
Mass	< 45 kg	N/A
Power Generated at 0.6 $\frac{m}{s}$ (Venus)	9 W	Max: 10.3 W
Efficiency	≥ 40 %	Max: 46.1%
Wind Speed Range (Venus)	(0.3 $\frac{m}{s}$ – 1.3 $\frac{m}{s}$)	In Progress
Budget	< $1,000	< $1,000

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Future Plans

- Continue to improve water testbed setup
  - Electrical/Mechanical braking system
- Optimize or redesign wind turbine blades
  - CFD and BEM simulation environments
- Future water testing
- More frequently available water testing environment

Thank you for your attention!

Any Questions?

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References

[1] Jet Propulsion Laboratory California Institute of Technology “Deadline Extended – Mechanical Maker Challenge: Mechanical Eye.” 31 March 2019 [Online]. Available: https://www.jpl.nasa.gov/edu/events/2018/9/26/mechanical-maker-challenge-mechanical-eye/

[2] Dr. J. Sauder, “Requirements Summary,” Los Angeles, 2020

[3] J. Manwell, J. McGowan and A. Rogers, *Wind Energy Explained*. Chichester: Wiley, 2009.